

Dept. of Communication Tech. Engineering

Fourth Stage



Control System

Al- Farahidi University

2023-2024

Lec.8

Time Response Analysis

Assistant lecturer

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Introduction

- In time-domain analysis the response of a dynamic system to an input is expressed as a function of time.
 - It is possible to compute the time response of a system if the nature of input and the mathematical model of the system are known.
 - Usually, the input signals to control systems are not known fully ahead of time.
 - For example, in a radar tracking system, the position and the speed of the target to be tracked may vary in a random fashion.
 - It is therefore difficult to express the actual input signals mathematically by simple equations.
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Standard Test Signals

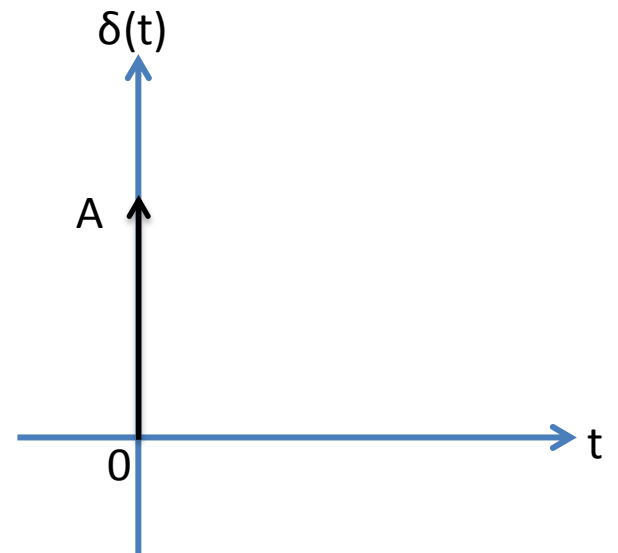
اشارات الفحص

- The characteristics of actual input signals are a sudden shock, a sudden change, a constant velocity, and constant acceleration.
- The dynamic behavior of a system is therefore judged and compared under application of standard test signals – an impulse, a step, a constant velocity, and constant acceleration.
- Another standard signal of great importance is a sinusoidal signal.

Standard Test Signals

- Impulse signal إشارة النبضة
 - The impulse signal imitate the sudden shock characteristic of actual input signal.

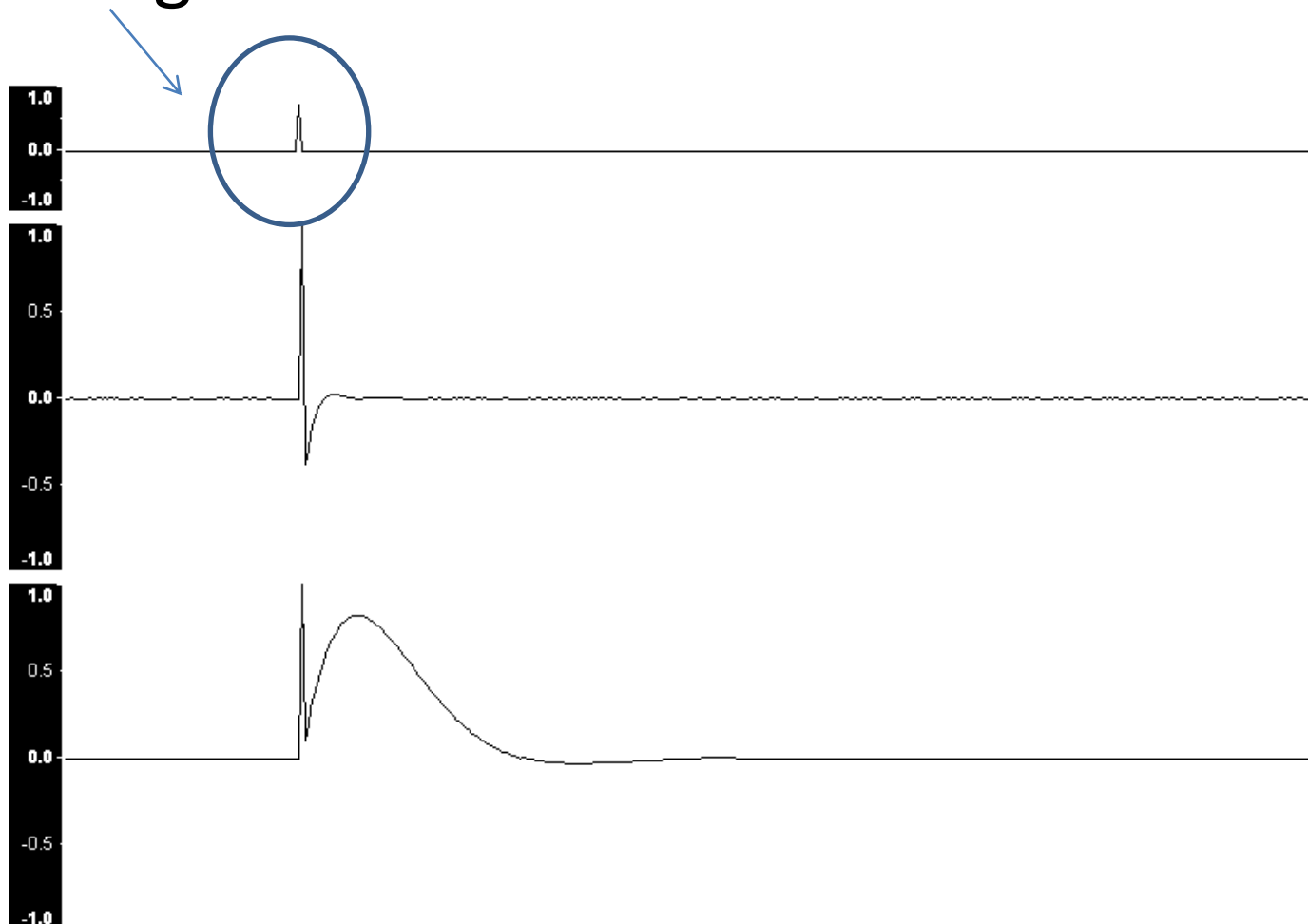
$$\delta(t) = \begin{cases} A & t = 0 \\ 0 & t \neq 0 \end{cases}$$



- If $A=1$, the impulse signal is called unit impulse signal.

Standard Test Signals

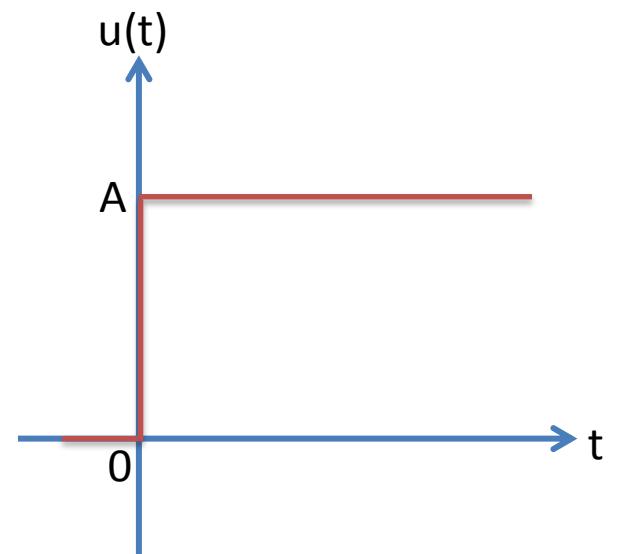
- Impulse signal



Standard Test Signals

- Step signal إشارة الخطوة
 - The step signal imitate the sudden change characteristic of actual input signal.

$$u(t) = \begin{cases} A & t \geq 0 \\ 0 & t < 0 \end{cases}$$

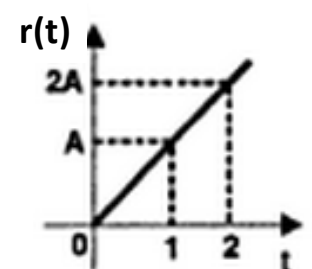
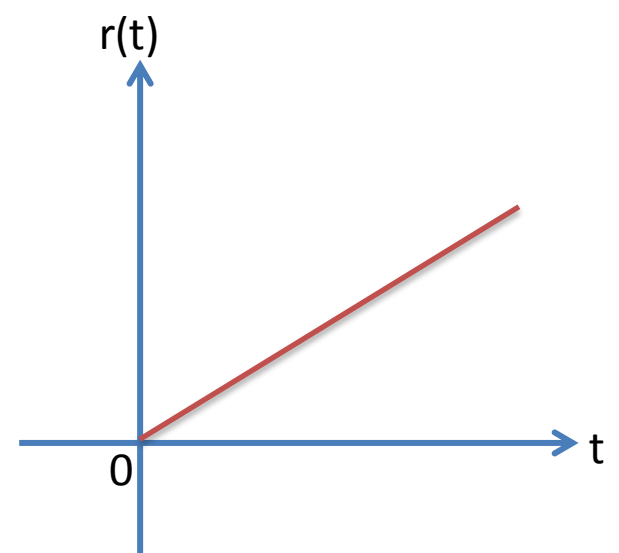


- If $A=1$, the step signal is called unit step signal

Standard Test Signals

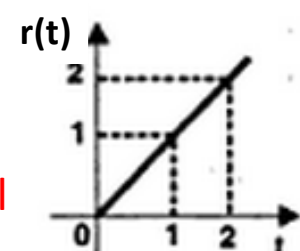
- Ramp signal إشارة المنحدر
 - The ramp signal imitate the constant velocity characteristic of actual input signal.

$$r(t) = \begin{cases} At & t \geq 0 \\ 0 & t < 0 \end{cases}$$



ramp signal with slope A

- If $A=1$, the ramp signal is called unit ramp signal



unit ramp signal

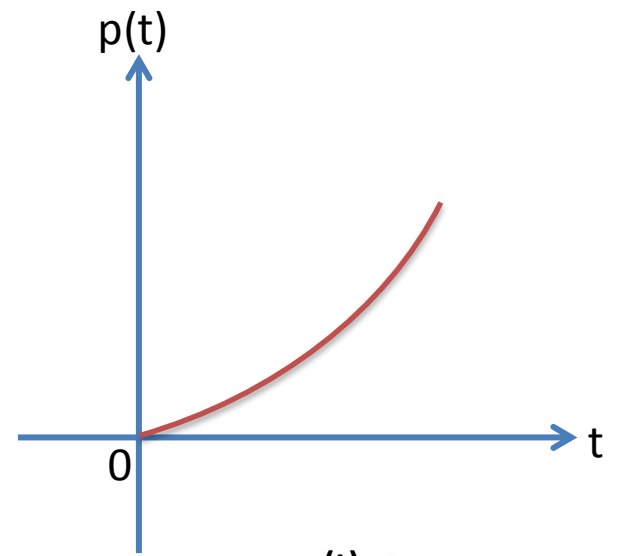
Standard Test Signals

- Parabolic signal إشارة القطع المكافئ

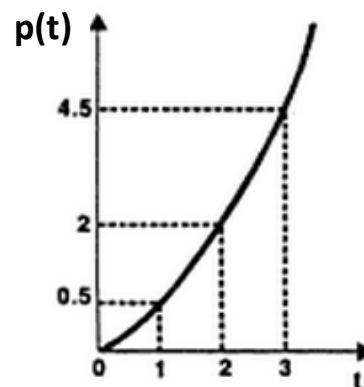
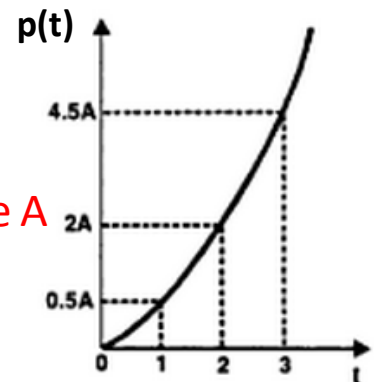
– The parabolic signal imitate the constant acceleration characteristic of actual input signal.

$$p(t) = \begin{cases} \frac{At^2}{2} & t \geq 0 \\ 0 & t < 0 \end{cases}$$

– If $A=1$, the parabolic signal is called unit parabolic signal.



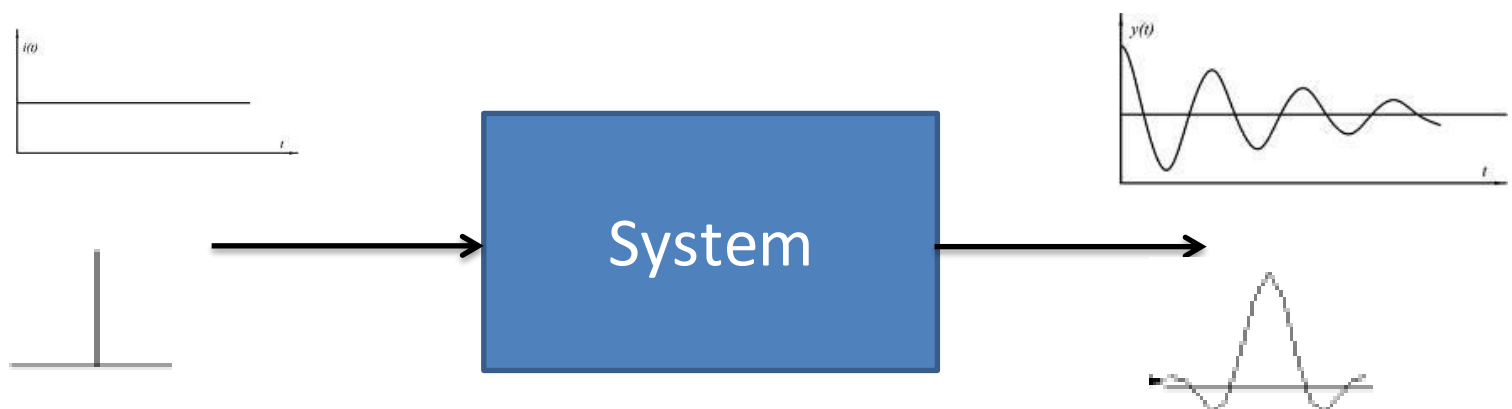
parabolic signal with slope A



Unit parabolic signal

Time Response of Control Systems

- Time response of a dynamic system response to an input expressed as a function of time.



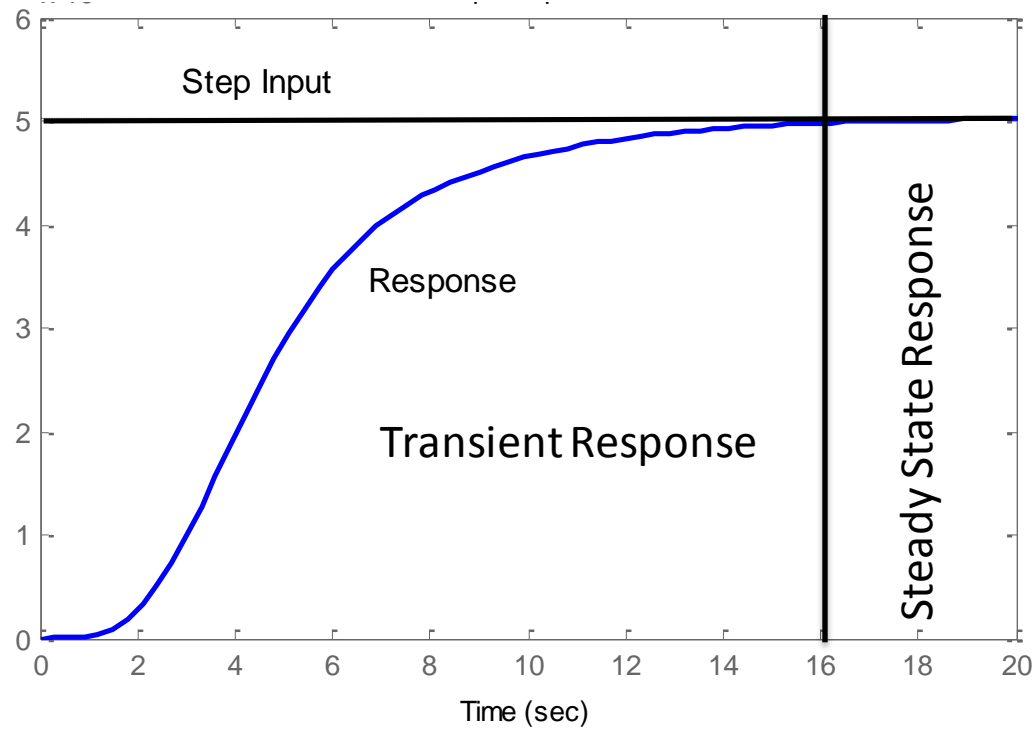
- The time response of any system has two components
 - Transient response
 - Steady-state response.

Time Response of Control Systems

الاداء الزمني لمنظومات السيطرة

- When the response of the system is changed from rest or equilibrium it takes some time to settle down.
- Transient response is the response of a system from rest or equilibrium to steady state.

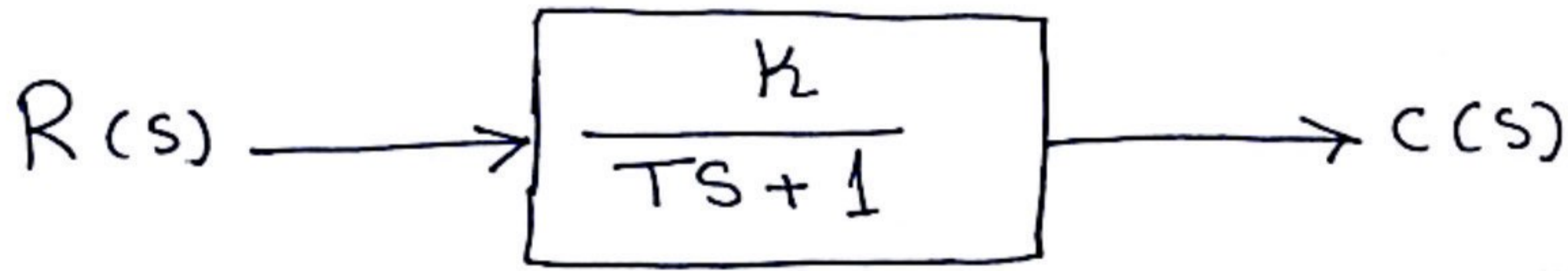
- The response of the system after the transient response is called steady state response.



Time Response Analysis

التحليل الزمني لمنظومات السيطرة

1- First Order System



$$\frac{C(s)}{R(s)} = \frac{k}{TS + 1}$$

$R(s)$ = input signal

$R(s) : \sim$

Impulse input
 $R(s) = A$
unit impulse i/p $\rightarrow R(s) = 1$

Step Input
 $R(s) = \frac{A}{s}$
unit step i/p $\rightarrow R(s) = \frac{1}{s}$

Ramp Input

$$R(s) = \frac{A}{s^2}$$

unit ramp i/p $\rightarrow R(s) = \frac{1}{s^2}$

ex: first order system with unit step input :~

$$\frac{C(s)}{R(s)} = \frac{k}{Ts+1}$$

$$\therefore C(s) = \frac{k}{Ts+1} * R(s)$$

$$= \frac{k}{Ts+1} * \frac{1}{s}$$

ex: first order system with ramp input :~

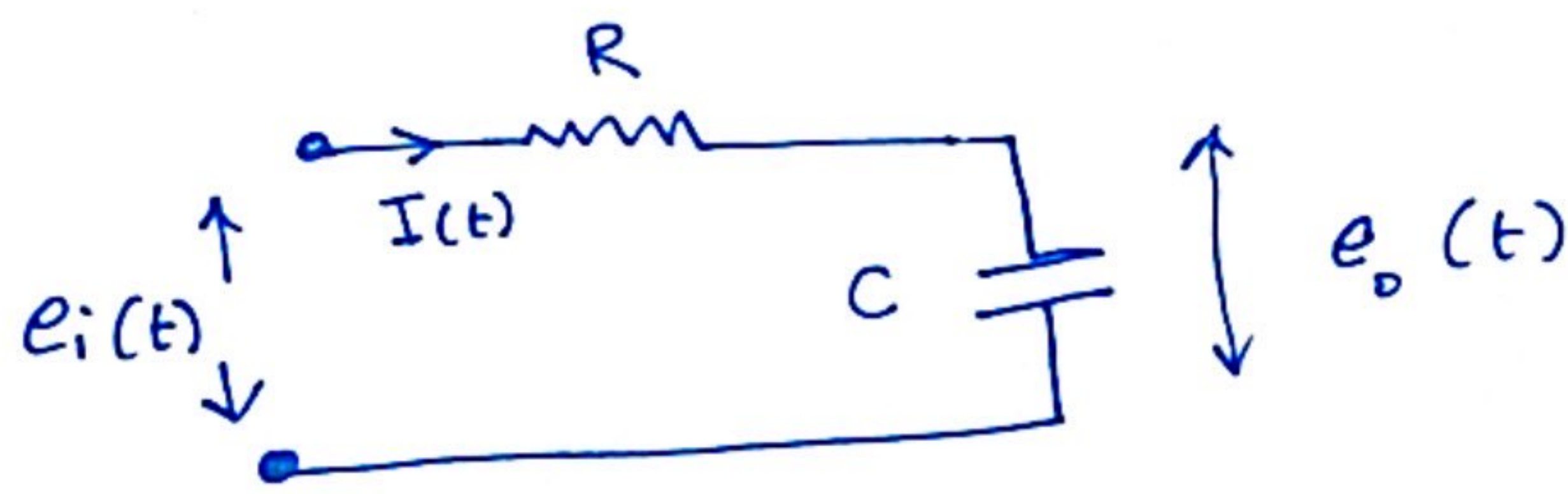
$$\frac{C(s)}{R(s)} = \frac{k}{Ts+1}$$

$$\therefore C(s) = \frac{k}{Ts+1} * R(s)$$

$$= \frac{k}{Ts+1} * \frac{A}{s^2} \rightarrow \text{or } a$$

ex: Check if the following System represent a first order system, then find $[k, T, a]$, if:

Sol



$$e_i(t) = 5 \text{ V}$$

Sol

$$\frac{e_o(s)}{e_i(s)} = \frac{I(s) * \frac{1}{Cs}}{[I(s) * R] + [I(s) * \frac{1}{Cs}]}$$

$$= \frac{I(s) * \frac{1}{Cs}}{I(s) [R + \frac{1}{Cs}]}$$

$$= \left[\frac{1/Cs}{R + 1/Cs} \right] * \frac{Cs}{Cs}$$

$$\therefore \frac{e_o(s)}{e_i(s)} = \frac{1}{Rcs + 1}$$

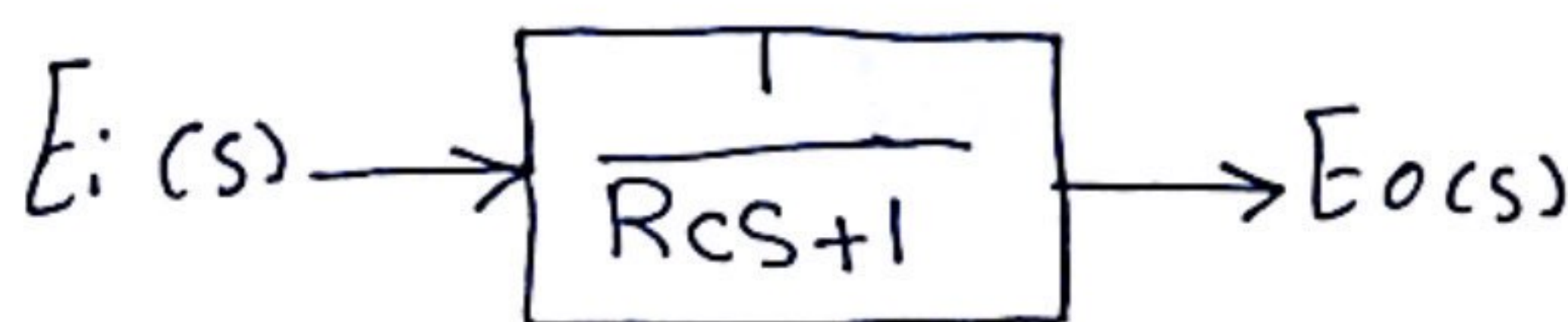
note: ~

$$\therefore e_o(s) = \frac{1}{Rcs + 1} * e_i(s)$$

$$e_i(t) = 5$$

↓ taking Laplace

$$\therefore e_i(s) = \frac{5}{s}$$



$$\therefore E_o(s) = \frac{1}{Rcs + 1} * \frac{5}{s}$$

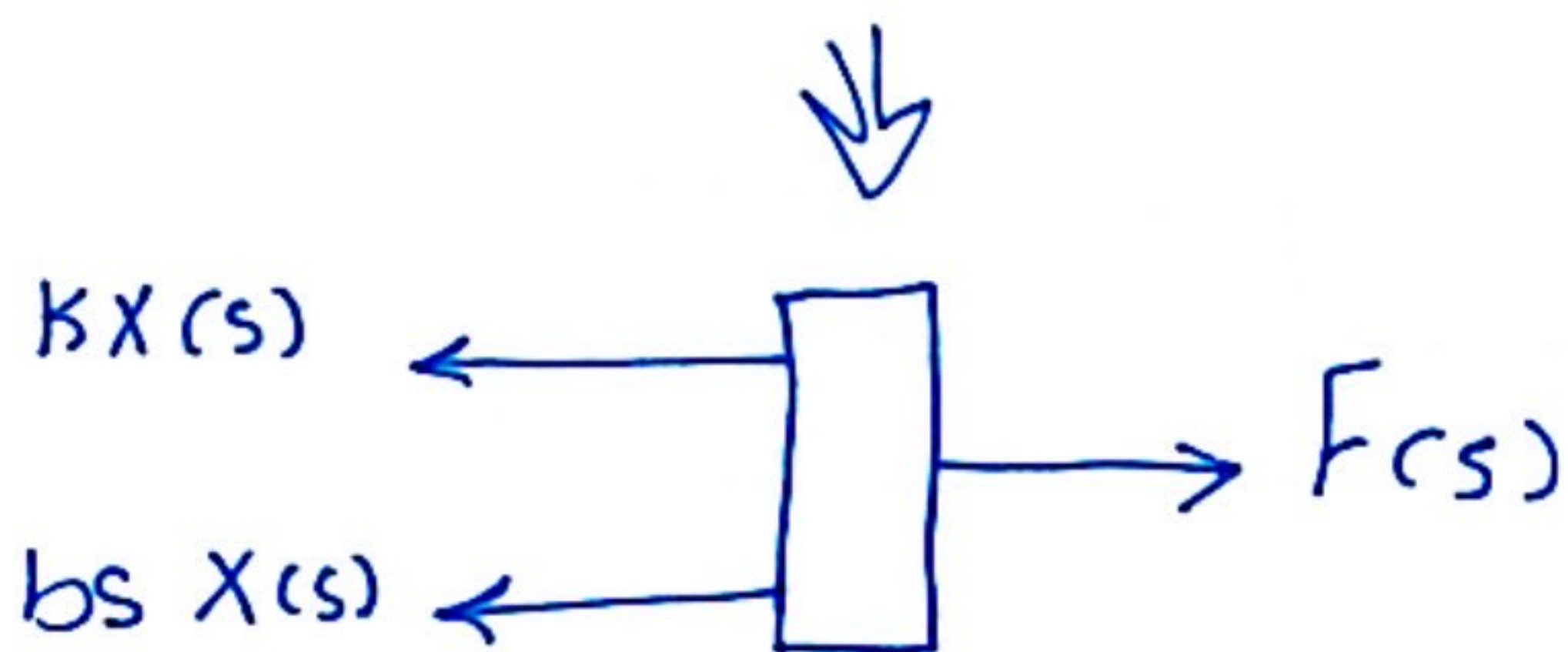
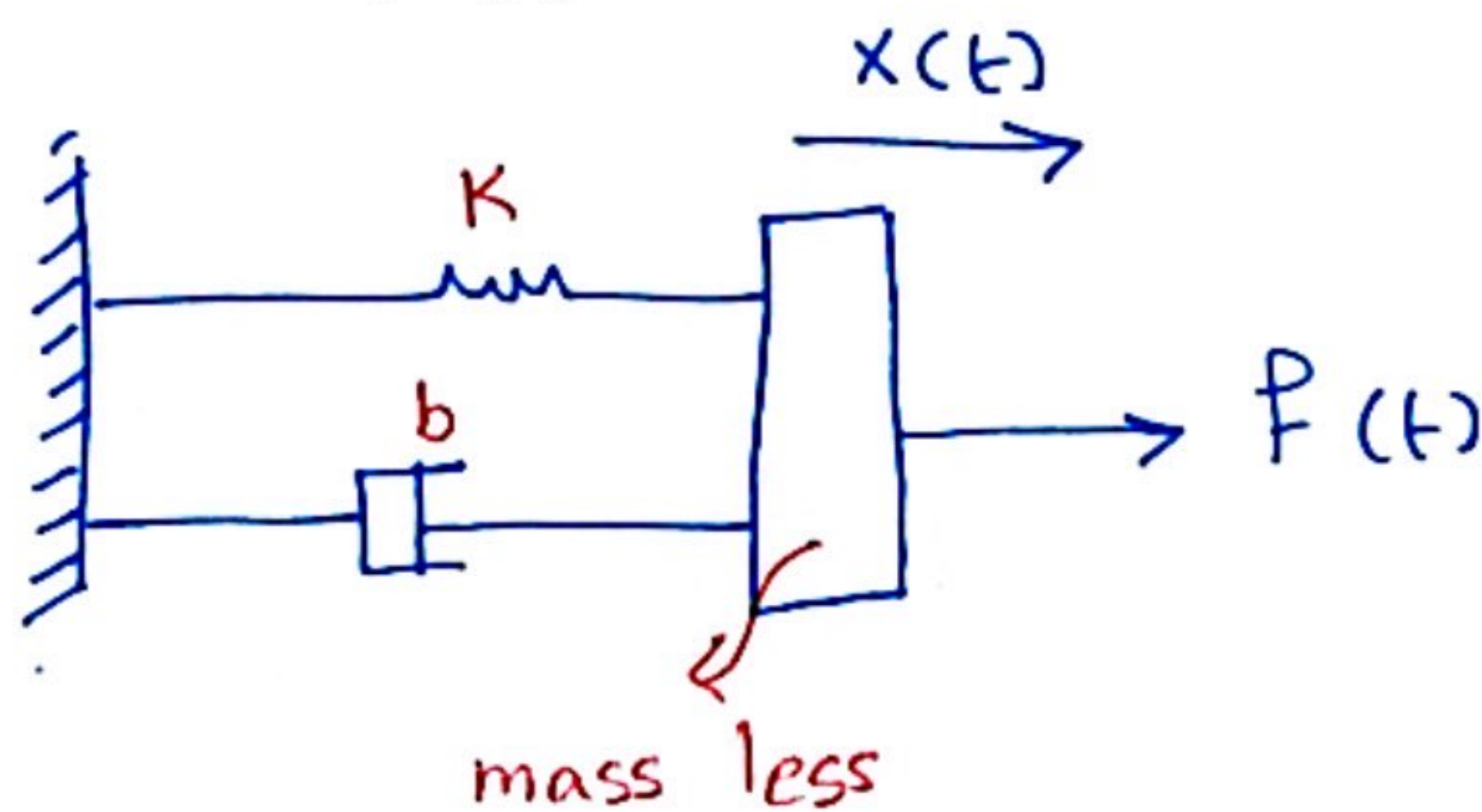
$$\boxed{\therefore k=1, T=RC, a=5}$$

ex: Find $\frac{X(s)}{F(s)}$, if $f(t) = 50t \text{ N}$ نيوتن

Laplace:

$$F(s) = \frac{50}{s^2}$$

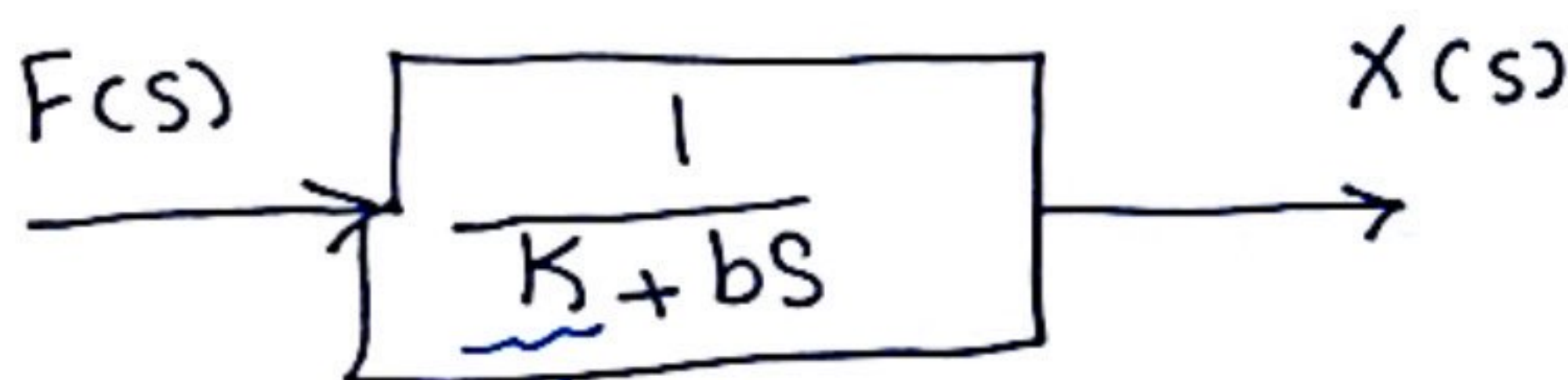
Ramp i/p



$$KX(s) + bS X(s) = F(s)$$

$$X(s) [K + bS] = F(s)$$

$$\therefore \frac{X(s)}{F(s)} = \frac{1}{K + bS}$$



بجواب النيوتن 1
في مقامه K

$$\therefore \frac{X(s)}{F(s)} = \frac{1/K}{1 + \frac{b}{K}S}$$

$$\therefore X(s) = \frac{1/K}{\frac{b}{K}S + 1} * F(s)$$

$$= \left[\frac{1/K}{\frac{b}{K}S + 1} * \frac{50}{s^2} \right]$$

$$\therefore K = \frac{1}{k}$$

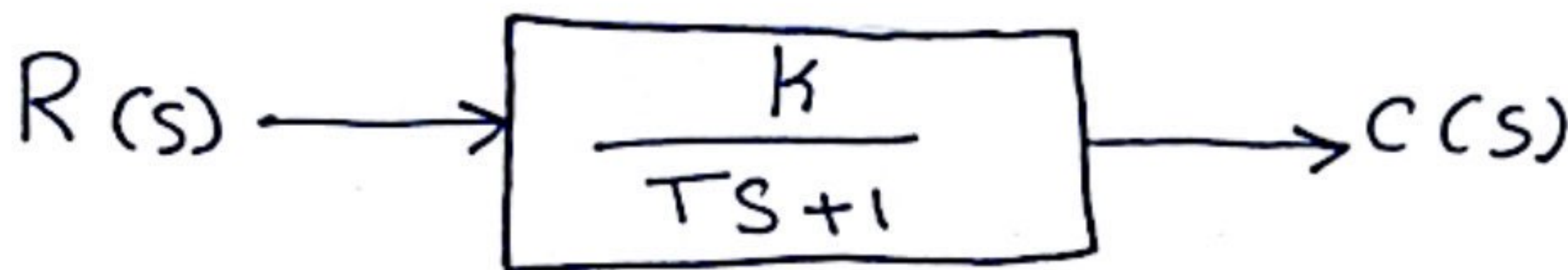
$$T = \frac{b}{k}$$

$$a = 50$$

ex: First Order System with Step Input Response:

sol

$$R(s) = \frac{a}{s}$$



$$\frac{C(s)}{R(s)} = \frac{k}{TS+1}$$

$$\therefore C(s) = \frac{k}{TS+1} * R(s)$$

$$\therefore C(s) = \frac{k}{TS+1} * \frac{a}{s}$$

الآن : لكي نقوم بعملية التحليل

الزمني [Time Response Analysis]

يجب أن نحول الدالة إلى

Time domain

∴ يجب استخدام Laplace inverse

في مثل هذه الحالات
نستخدم طريقة Partial
Laplace inverse لايجاد

now, finding $c(t)$:

$$C(s) = \frac{ka}{(TS+1)s} = \frac{A}{TS+1} + \frac{B}{s}$$

$$A = \left[\frac{ka}{s} \right]_{s=-\frac{1}{T}}$$

$$\therefore \boxed{A = -kaT}$$

نغوض المقامات بصفر
ونجد الحدود أو الجذور

$$[TS+1=0 \Rightarrow TS=-1 \Rightarrow s=-\frac{1}{T}]$$



$$B = \left[\frac{ka}{Ts+1} \right]_{s=0} \Rightarrow \boxed{B = ka}$$

$$\therefore C(s) = \frac{\overset{A}{-kaT}}{Ts+1} + \frac{\overset{B}{ka}}{s}$$

$$\therefore C(s) = \frac{-ka}{s + \frac{1}{T}} + \frac{ka}{s}$$

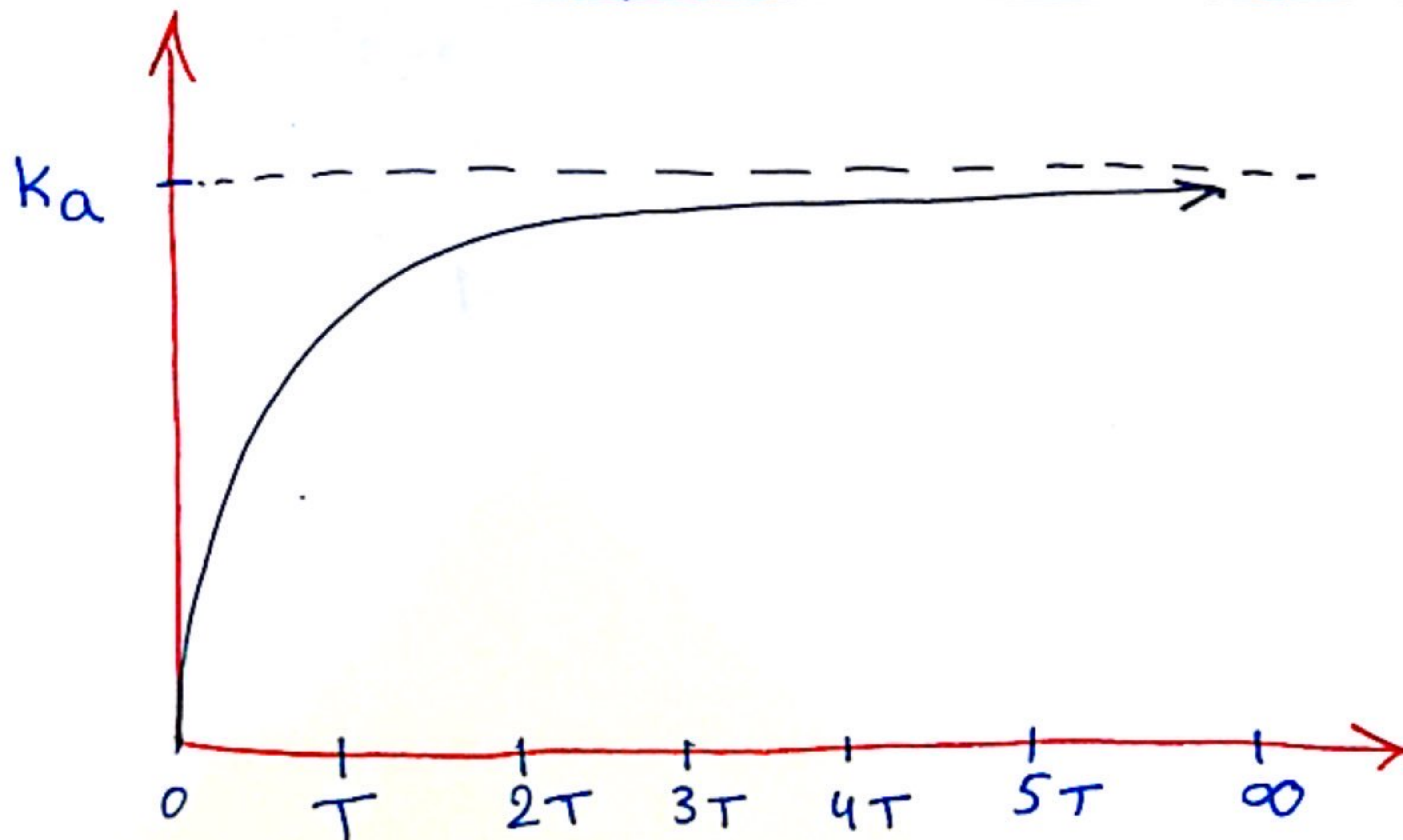
$$e^{-at} \xrightarrow{\mathcal{L}^{-1}} \frac{1}{s+a}$$

الآن : بعد التحويل
من S domain إلى
Time domain

$$\therefore C(t) = -ka e^{-\frac{1}{T} \cdot t} + ka$$

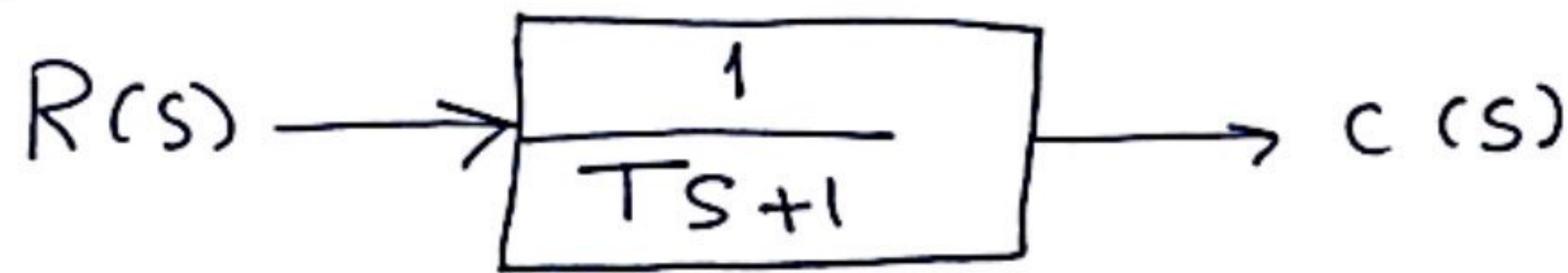
$$\boxed{\therefore C(t) = ka \left[1 - e^{-\frac{1}{T} \cdot t} \right]}$$

t	0	T	2T	3T	4T	5T	∞
C(t)	0	0.63 ka	0.86 ka	0.95 ka	0.98 ka	0.99 ka	ka



ex: First Order System with unit step input:

Sol)



$$R(s) = \frac{1}{s}$$

Sol) From the block diagram we found that:

$$K=1, T=T, a=1, \text{ unit step i/p}$$

$$\therefore C(s) = \frac{1}{TS+1} * R(s)$$

$$= \frac{1}{TS+1} * \frac{1}{s} = \frac{1}{s(Ts+1)}$$

$$= \frac{A}{Ts+1} + \frac{B}{s}$$

$$\therefore A = \left[\frac{1}{s} \right]_{s=-\frac{1}{T}} \rightarrow \boxed{A = -T}$$

$$B = \left[\frac{1}{Ts+1} \right]_{s=0} \rightarrow \boxed{B = 1}$$

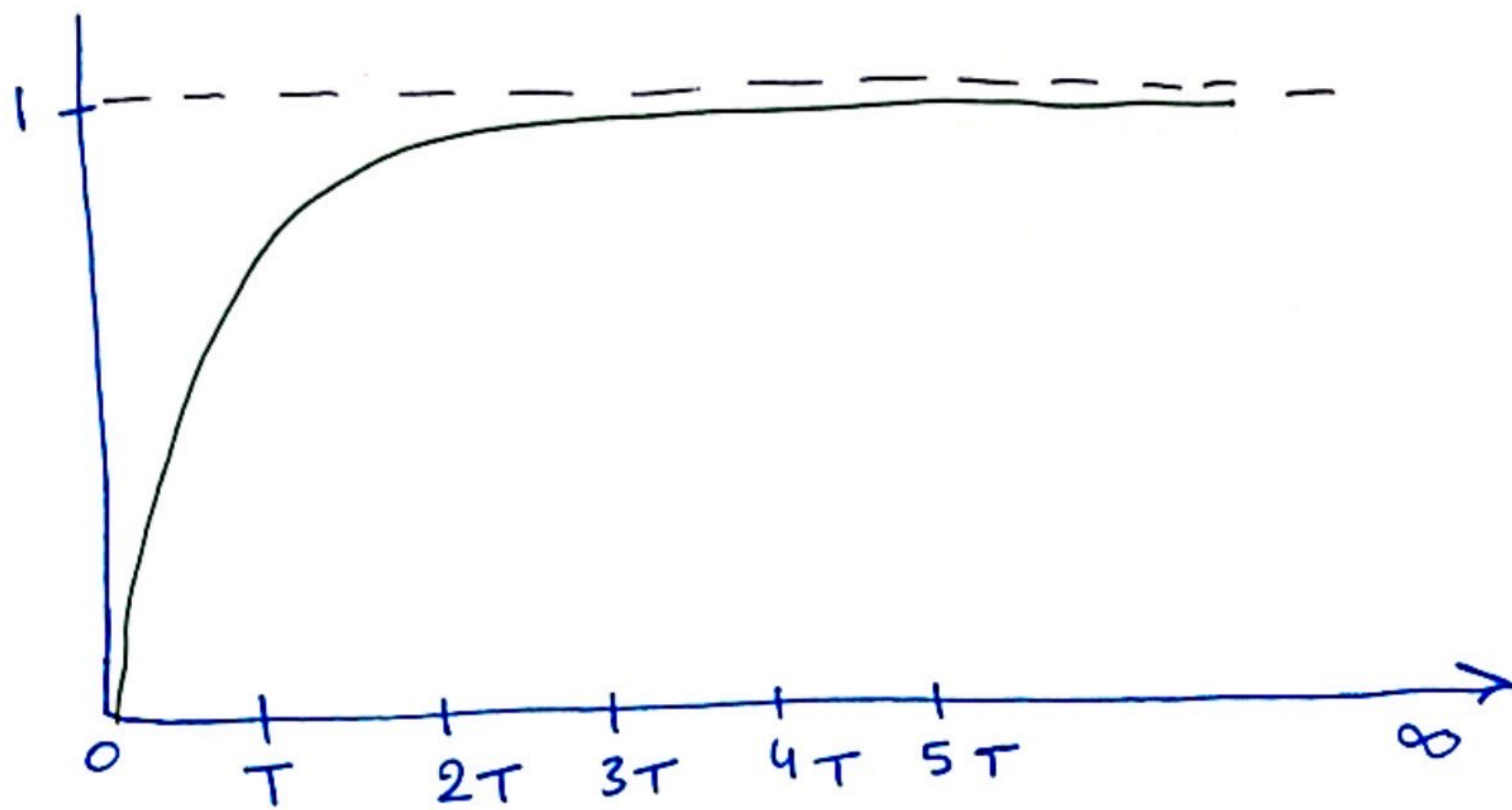
$$\therefore C(s) = \frac{-T}{Ts+1} + \frac{1}{s}$$

$$= \frac{-1}{s + \frac{1}{T}} + \frac{1}{s}$$

$$\therefore C(t) = -e^{-\frac{1}{T} \cdot t} + 1$$

$$= 1 - e^{-\frac{1}{T} \cdot t}$$

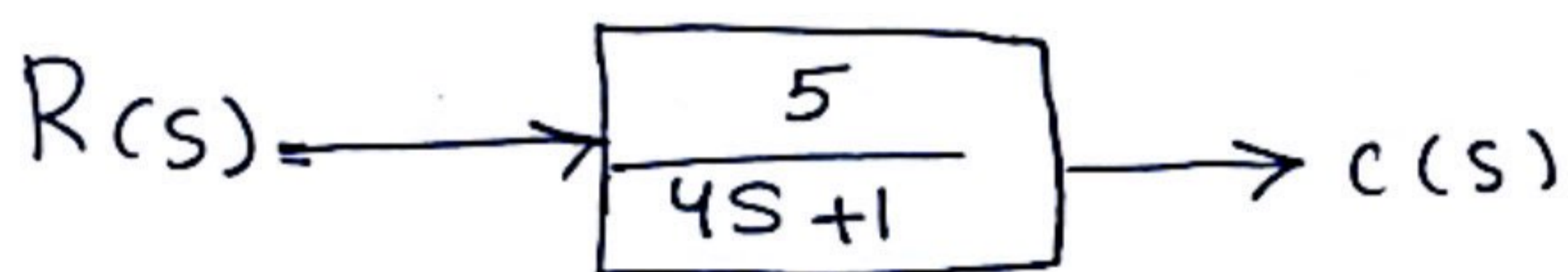




EX: First Order System with unit Impulse Response

$$R(t) = a$$

$$\therefore R(s) = 1$$



$K=5$, $T=4$, $a=1$, unit impulse input.

$$\frac{C(s)}{R(s)} = \frac{5}{4s+1} \longrightarrow C(s) = \frac{5}{4s+1} * R(s)$$

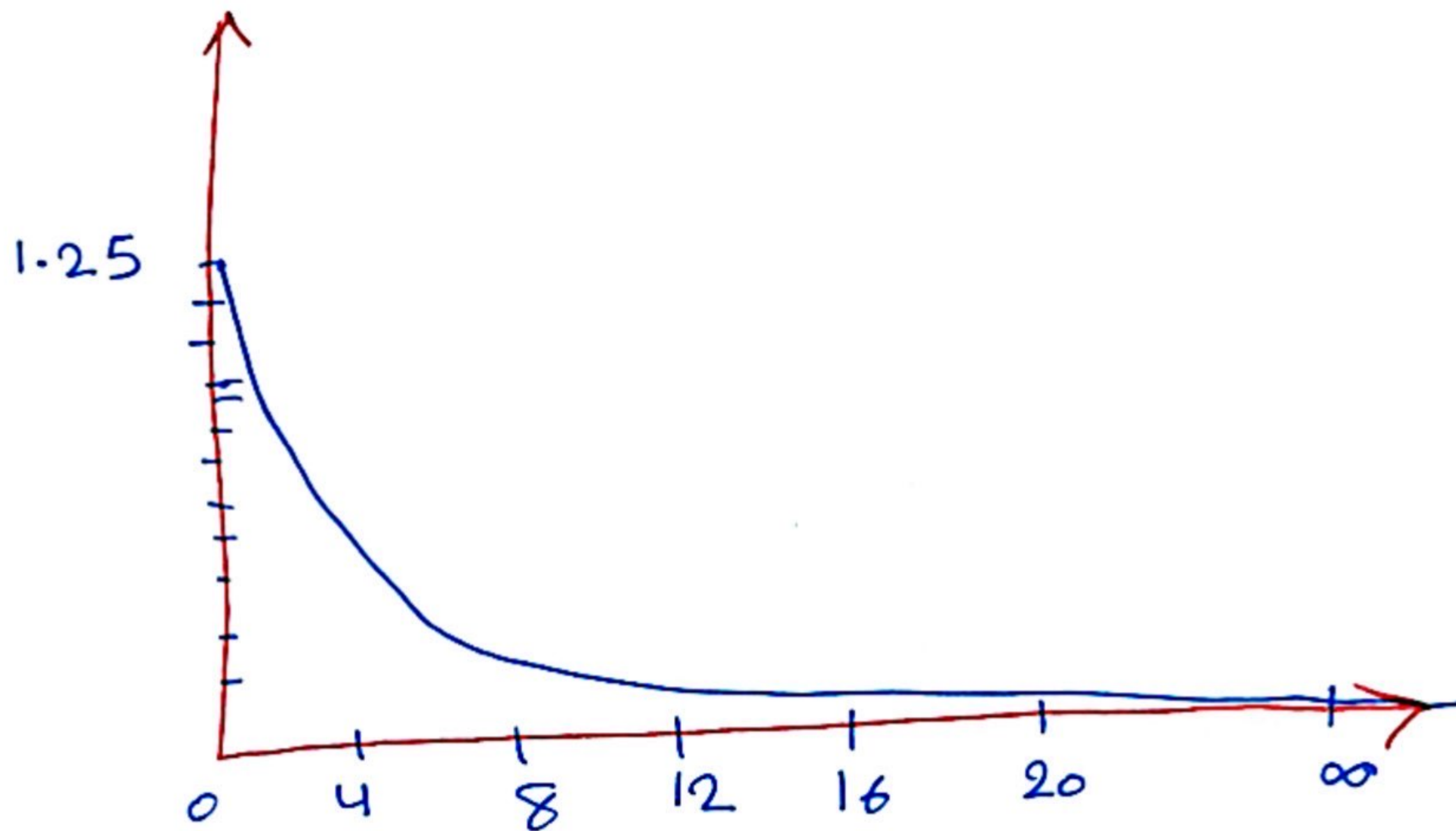
$$\therefore C(s) = \frac{5}{4s+1} * 1 \leftarrow \text{i/p}$$

$$= \frac{5/4}{s + 1/4}$$

$$\therefore C(t) = \frac{5}{4} e^{-1/4 t}$$



		0	T	$2T$	$3T$	$4T$	$5T$	
t	0	4	8	12	16	20	∞	
$c(t)$	$5/4$	0.46	0.17	0.06	0.02	8.4×10^{-3}	0	



H.W

First Order System with Ramp input
response :

$$R(s) = \frac{a}{s^2}$$



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Fourth Stage



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Lec.9

Second Order System

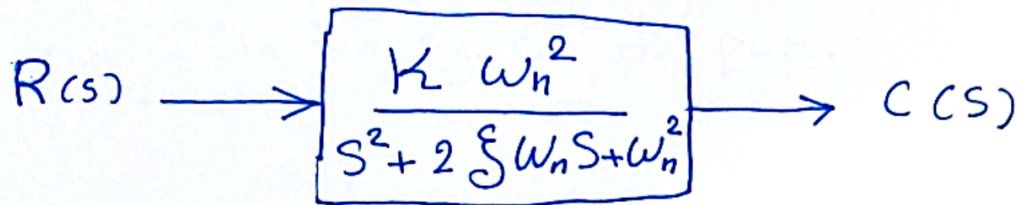
Assistant lecturer

Yasameen Hameed Al-aarajy

Time Response Analysis

2. Second-Order System

A general second-order system is characterized by the following transfer function



$$\frac{C(s)}{R(s)} = \frac{K \cdot \omega_n^2}{s^2 + 2 \xi \omega_n s + \omega_n^2}$$

$$\therefore C(s) = \frac{K \cdot \omega_n^2}{s^2 + 2 \xi \omega_n s + \omega_n^2} * R(s)$$

The characteristic equation for the system is:

$$s^2 + 2 \xi \omega_n s + \omega_n^2 = 0$$

\therefore its roots [poles] are given by:

$$P_{1,2} = -\xi \omega_n \pm \omega_n \sqrt{\xi^2 - 1}$$

where

ω_n = Un-damped natural frequency.

ξ = Damping Ratio.

ω_n = un-damped natural frequency of the second order system. [which is the frequency of oscillation of the system without damping].

ξ = damping ratio of the second order system, which is a measure of the degree of resistance to change in the system output.

EX: Determine the un-damped natural frequency and damping ratio of the following system: ~

$$\frac{C(s)}{R(s)} = \frac{4}{s^2 + 2s + 4}$$

Sol)

$$\frac{C(s)}{R(s)} = \frac{K \omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

by comparing:

$$* \omega_n^2 = 4 \rightarrow \omega_n = 2 \text{ rad/sec}$$

$$* K \omega_n^2 = 4 \rightarrow K = 1$$

$$* 2\xi\omega_n = 2 \rightarrow \xi = \frac{2}{2\omega_n} = \frac{1}{\omega_n} = \frac{1}{2} = 0.5$$

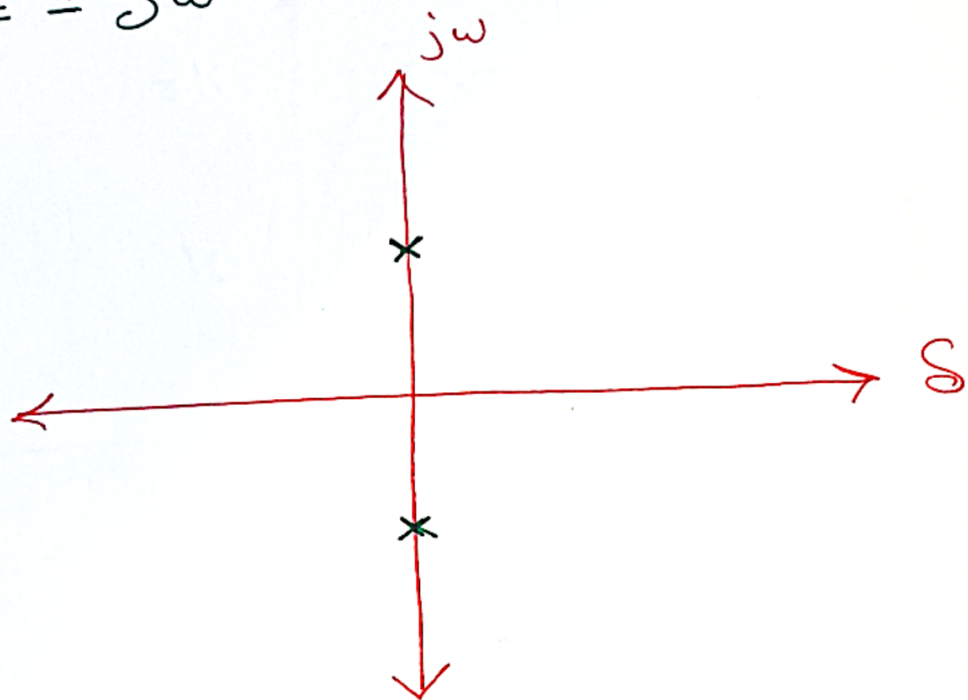
According to the value of ξ , a second order system can be set into one of the four categories:

1- Un-Damped \approx

when the system has two imaginary poles.

$$[\xi = 0]$$

$$P_{1,2} = \pm j\omega$$



$$G(s) = \frac{K \omega_n^2}{s^2 + \omega_n^2}$$

* For step response :

$$C(s) = G(s) * R(s)$$

$$= \frac{K \omega_n^2}{s^2 + \omega_n^2} * \frac{1}{s}$$

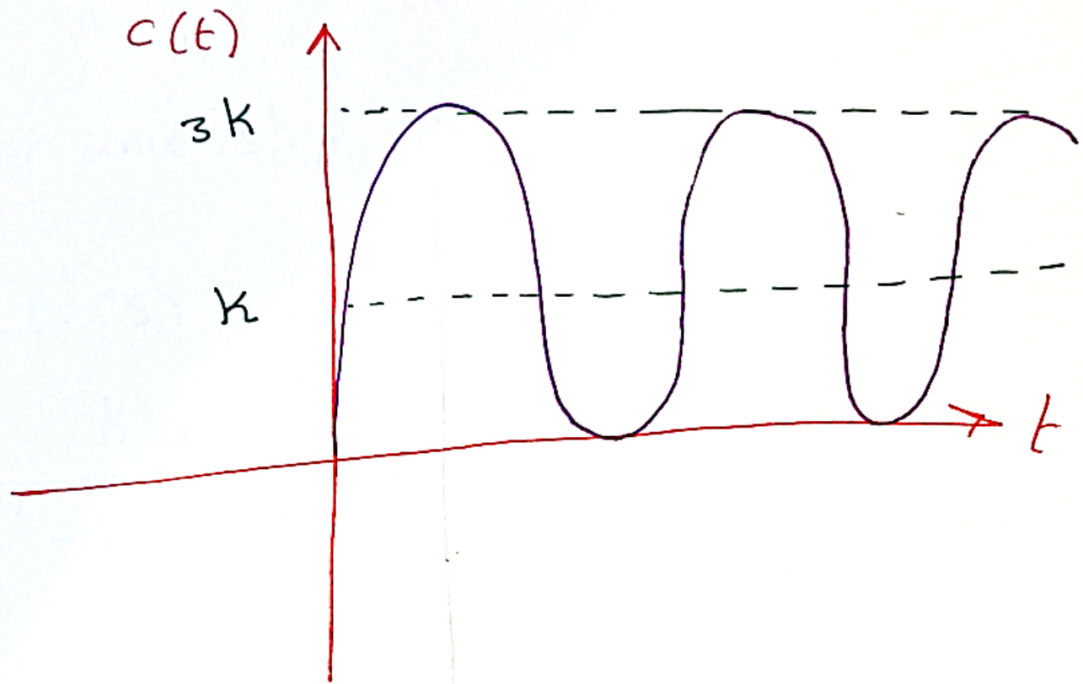
← by Partial fraction



$$\therefore C(s) = k \left[\frac{1}{s} - \frac{s}{s^2 + \omega_n^2} \right]$$

by taking Laplace Inverse :

$$\therefore C(t) = k [1 - \cos \omega_n t]$$

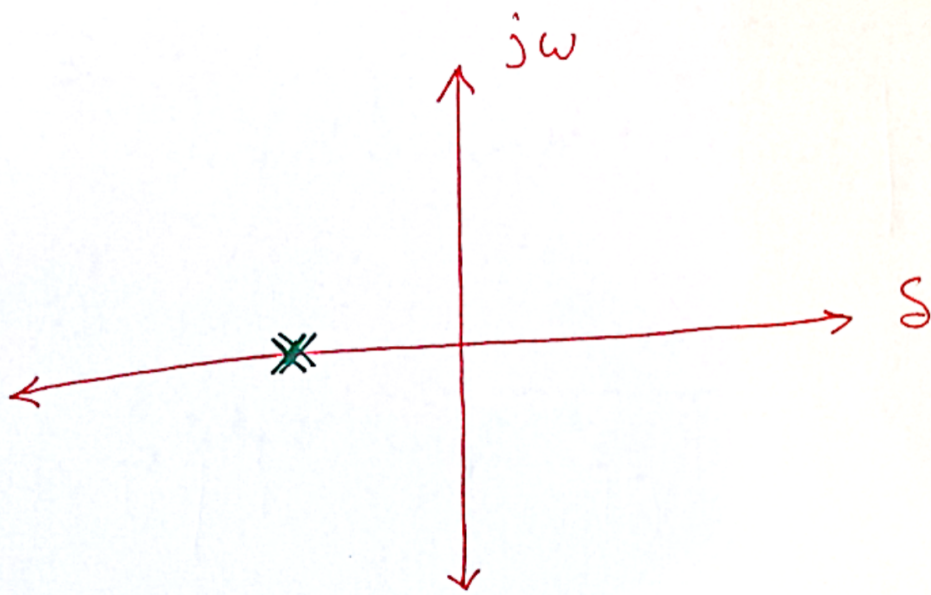


2. Critically Damped or
when the system has two real but equal poles.

$$[\zeta = 1]$$

$$P_{1,2} = -\zeta \omega_n \pm \sqrt{\zeta^2 - 1} \cdot \omega_n$$
$$= -\omega_n$$





* For unit step response:

$$C(s) = G(s) * R(s)$$

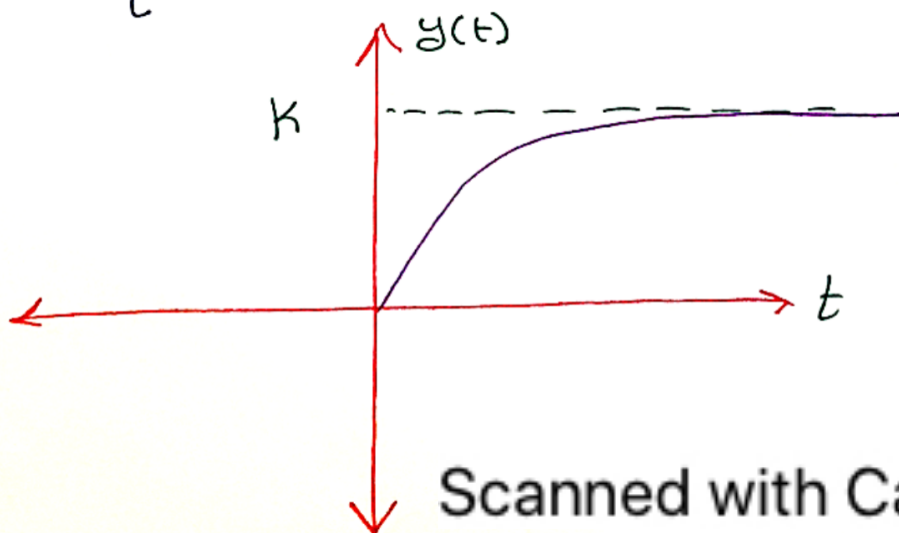
$$= \frac{k \cdot \omega_n^2}{s^2 + 2\omega_n s + \omega_n^2} * \frac{1}{s}$$

$$= k \left[\frac{1}{s} - \frac{\omega_n}{(s + \omega_n)^2} - \frac{1}{s + \omega_n} \right]$$

by Partial fraction

by Taking Laplace inverse:

$$\therefore c(t) = k \left[1 - e^{-\omega_n t} - \omega_n t \cdot e^{-\omega_n t} \right]$$

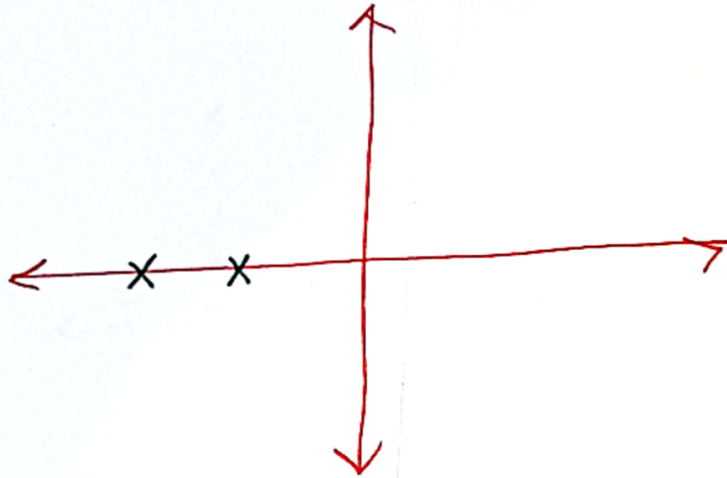


3. Over Damped \approx

when the system has two real distinct poles.

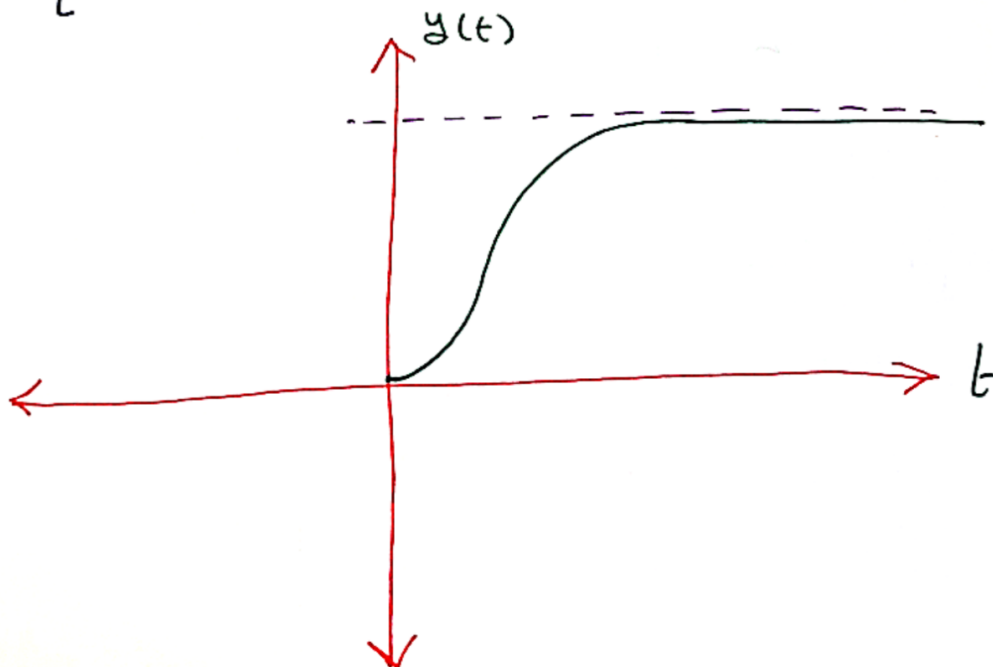
$$[\zeta > 1]$$

$$P_{1,2} = -\zeta \omega_n \pm \omega_n \sqrt{\zeta^2 - 1}$$



For unit step input:

$$c(t) = K \left[1 - k_1 e^{(-\zeta \omega_n + \omega_n \sqrt{\zeta^2 - 1})t} - k_2 e^{(-\zeta \omega_n - \omega_n \sqrt{\zeta^2 - 1})t} \right]$$

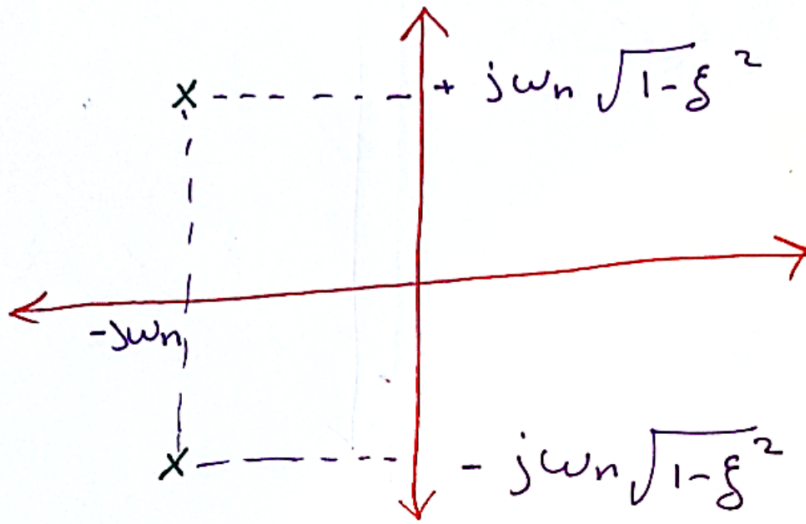


4- Under-Damped or

when the system has two complex conjugate poles

$$[0 < \xi < 1]$$

$$P_{1/2} = -\xi \omega_n \pm j \omega_n \sqrt{1 - \xi^2}$$



* For unit step input.

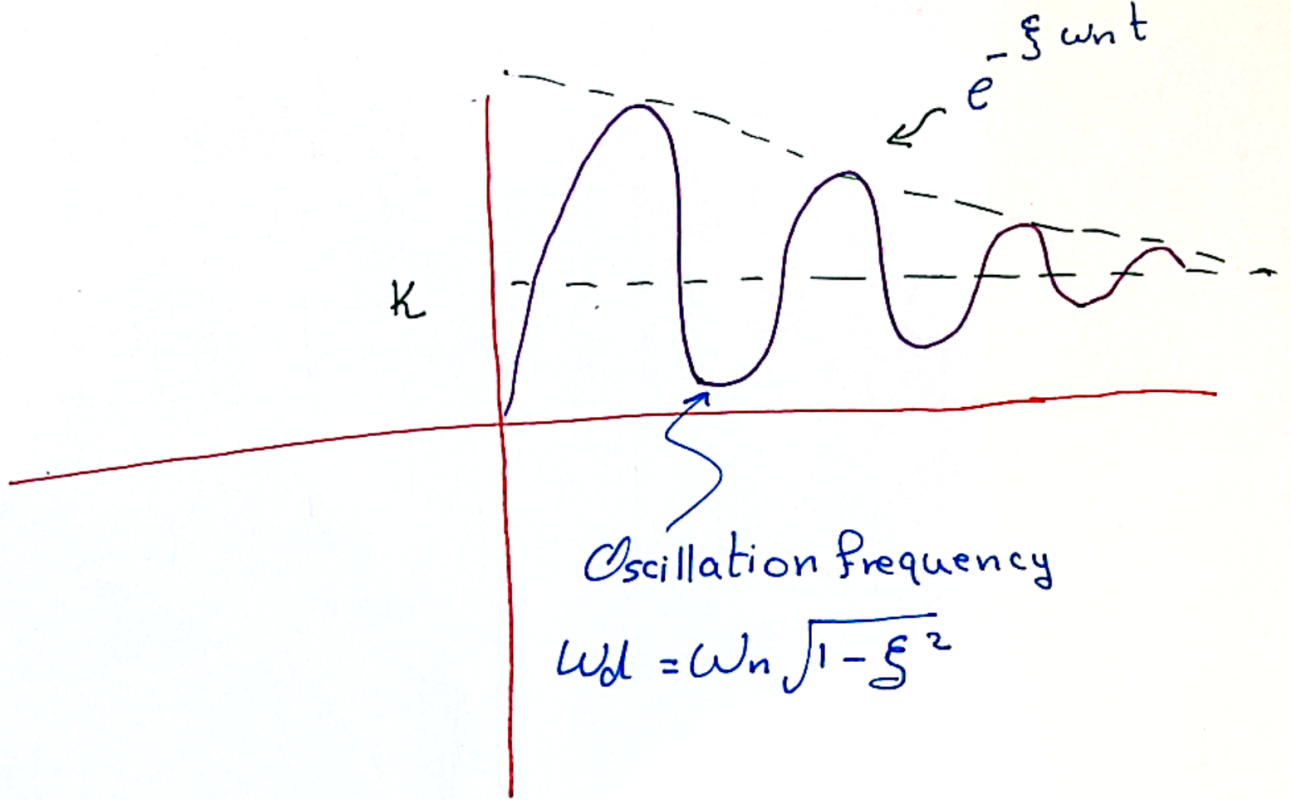
$$C(s) = \frac{K \omega_n^2}{s^2 + 2\xi \omega_n s + \omega_n^2} * \frac{1}{s}$$

$$C(t) = K \left[1 - \frac{1}{\sqrt{1 - \xi^2}} e^{-\xi \omega_n t} \cdot \cos \left[\omega_n \sqrt{1 - \xi^2} t - \phi \right] \right]$$

where:

$$\phi = \tan^{-1} \left[\frac{\xi}{\sqrt{1 - \xi^2}} \right]$$





$$\omega_d = \omega_n \sqrt{1 - \xi^2}$$